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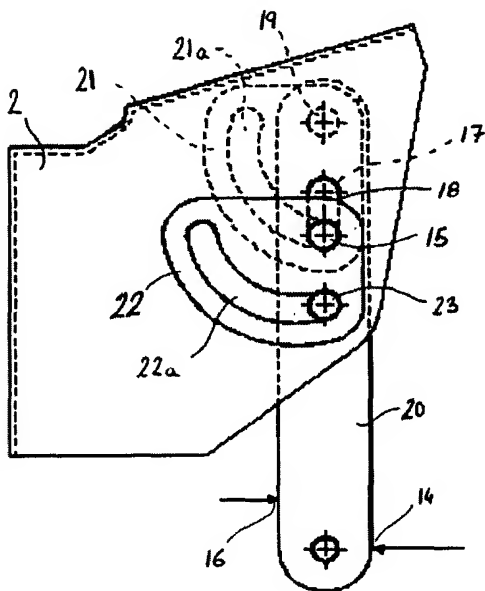
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(54) Title: **RATIO REGULATION MECHANISM FOR AN ACTION LEVER**



(57) **Abstract:** The invention relates to a ratio regulating mechanism for an manually actuated action lever, like a hand brake lever or a pedal, in particular for the use in a motor vehicle, which comprises a mounting (2), an action lever arm (20) and a rotational shaft (15) for rotably mounting of the action lever arm (20) at the mounting (2), wherein the rotational shaft (15) is relocatably mounted at the action lever arm (20) and relocatably mounted at the mounting (2). Further, the ratio regulating mechanism comprises a first adjustment means (19, 21) for the relocation of the rotational shaft (15) in relation to the action lever arm (20) and a second adjustment means (22, 23) for the relocation of the rotational shaft (15) in relation to the mounting (2).

WO 2004/048169 A1

Ratio regulation mechanism for an action lever

1. Technical field

5 The present invention relates to a ratio regulating mechanism for an action lever for changing the forces and ways transmitted by the lever. The present invention in particular relates to a ratio regulating mechanism for an manually actuated action lever of a motor vehicle, like a hand brake lever or a pedal with adjustable lever ratio.

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2. Prior art

In many fields of mechanics, e.g. in mechanical engineering or motor-vehicle construction, there is the problem of adjusting a lever in that various actuation forces and actuation ways are possible. This may be caused by an action lever for
15 manual actuation having to be actuatable by persons of different strength. In this respect, it might be desirable to be able to adjust the required actuation force of a handbrake lever in a motor vehicle to the respective driver. A strong driver might want to have a short actuation way with a correspondingly high force whilst a weaker driver might desire a longer actuation way with lesser force.

20

The prior art describes manually actuated action levers that change their respective lever ratio during the actuation. The GB 1,092,314 describes a variable ratio lever mechanisms for a handbrake which uses a lever arm mechanics to produce in an initial portion of a stroke a large output movement with a low output force,
25 and in a final portion of the stroke a high output force. That means, the lever arm of the action lever changes during the stroke. The US 3,727,418 shows the opposite effect. Here, the initial portion of the stroke has a high output force and the final portion of the stroke a low output force. However these action levers do not show an adjustment of the lever ratio to the user.

- 2 -

Also in the case of pedals of motor vehicles, which are manually actuated by foot, it might be desirable to be able to adjust actuation forces and actuation ways.

5 Many different solutions for adjusting a pedal to a driver are described in the prior art. DE 199 23 697 A1 and DE 200 22 852 U1, for example, describe adjustable pedals which can be adjusted by the adjustment of the pedal to the respective driver. Therein, the lever arm of the pedal is extended in that the pedal is extended diagonally downwards in the direction towards the driver. From these solutions it
10 is obvious that the lever ratios change drastically after adjustment. In the adjustment for small drivers, the pedal has a long lever so that it has to be actuated with a long actuation way and correspondingly small force.

It is thus desirable in such cases that the lever ratios are maintained approximately
15 constant. It is desirable that small and large drivers have the same actuation way and the same actuation force in the actuation of the pedal.

Other pedals with a variable lever ratio are described in the US 3,646,830 or the JP 59 180 713 A wherein the lever ratio changes during the stroke of the pedal. In
20 these documents the lever ratio becomes larger while increasing the actuation, that means the stroke of the pedal.

Thus, the problem on which the invention is based consists of providing an improved manually actuated actuation lever wherein the lever ratio is adjustable to
25 the user. On the one hand, it is desired to have a regular actuation lever, wherein the lever ratio can be adjusted to the user. On the other hand, it is desired to have an action lever which provides the same lever ratio for all users after a geometric adjustment of the actuation lever to users of different size.

- 3 -

3. Summary of the invention

The invention on which the problem is based is according to the invention solved by a ratio regulating mechanism for an action lever according to patent claim 1.

- 5 The ratio regulating mechanism for an manually actuated action lever, in particular for the use in a motor vehicle, comprises a mounting, an action lever arm and a rotational shaft for rotably mounting of the action lever arm at the mounting, wherein the rotational shaft is relocatably mounted at the action lever arm and
10 relocatably mounted at the mounting. Further, the ratio regulating mechanism comprises a first adjustment means for the relocation of the rotational shaft in relation to the action lever arm and a second adjustment means for the relocation of the rotational shaft in relation to the mounting.

Therefore, the action lever arm rotates around the rotational shaft. Thus, it is possible to adjust the action lever in that the ratio between the lever arms of input
15 force F_1 and output force F_2 , which act on the arm, is variable, wherein the forces F_1 and F_2 are induced at the same positions before and after the relocation of the rotational shaft.

- 20 The ratio of the forces F_1 and F_2 is its quotient or its relation. In an action lever the ratio depends on the geometrical conditions of the force inducing points and rotation points. In our case the ratio is the quotient of the distances of the lines of forces of F_1 and F_2 to the rotational shaft, which defines the rotation axis of the action lever arm. By relocating the rotational shaft the distances to the lines of
25 forces of F_1 and F_2 are varied. Thereby, also the quotient of this distances varies. That leads to a variation of the lever ratio of the action lever.

Preferably, the first adjustment means and the second adjustment means are adjustable, so that during the relocation of the rotational shaft the position of the
30 action lever arm in relation to the mounting is maintained. Since the position of the action lever arm in relation to the fixed support is maintained, the force in-

- 4 -

ducing points on the action lever arm can be maintained. In particular the hand grip of e.g. a hand-brake lever is then always at the same position. Moreover, the position of e.g. a foot piece of a pedal is not changed.

- 5 Preferably, the relocation of the rotational shaft by said first adjustment means occurs in an opposite direction to the relocation of the rotational shaft by said second adjustment means.

10 Preferably, the first adjustment means and the second adjustment means are simultaneously actuated. Therefore, the action lever arm may not move with respect to the mounting during adjustment. Every intermediate adjustment position is directly obtained during adjustment.

15 Preferably, the first adjustment means is mounted at the action lever arm. Further preferred, the second adjustment means is mounted at the mounting. Thus, only the rotational shaft of the actuation lever is displaced with respect to the mounting and not the action lever itself. The first adjustment means will at least partially move together with the action lever arm if the action lever arm is actuated.

20 Preferably, the action lever arm comprises an elongated guide and the mounting comprises an elongated guide for the relocatable mounting of the rotational shaft. The position of the rotational shaft can therefore be changed in relation to the action lever arm as well as to the mounting.

25 Preferably, the first adjustment means comprises an arm shaft, which is supported in the action lever arm, and first cam plates, which are connected to the arm shaft and the rotational shaft, so that a relocation of the rotational shaft in relation to the action lever arm results from a rotation of the first cam plates.

30 Preferably, the second adjustment means comprises a support pin, which is connected to the mounting, and second cam plates, which are connected to the sup-

- 5 -

port pin and the rotational shaft, so that a relocation of the rotational shaft in relation to the mounting results from a rotation of the second cam plates.

5 Preferably, the first cam plates comprise first cam slots, through which the rotational shaft extends, and the second cam plates comprise second cam slot, through which the support pin extends.

10 Preferably, the rotational shaft is attached to the second cam plates and slideably arranged through the first cam slots, so that the rotational shaft is functional connected to both adjustment means.

15 Using elongated slots and cam plates is a very robust and low-maintenance possibility for displacing axes within elongated slots. Moreover, when two cam plates are connected to an axis with torsional connection, a very simple parallel guide can be provided. Moreover, it is advantageous that a cam plate is actuated by rotational movement which is very easily possible by an electric motor or a hand wheel.

20 The form of the cam slots is given by the length of the displacement/relocation of the rotational axis. The cam slots can have different shapes, preferably, however, the first and the second cam slots have the same shape and length. Herein, it is to be noted that the first and the second cam plates can be built into the actuation lever in different orientations despite their identical shapes.

25 Preferably, the first and the second cam plates are rotated by the same rotation angle during adjustment of the rotational shaft in order to maintain the position of the action lever arm in relation to the mounting.

30 The drive of the first and/or the second cam plates is preferably effected by means of an electric motor or manually.

- 6 -

The first and/or the second cam plates are preferably driven either by means of a toothed wheel gearing, a spindle gearing, a cam gearing, a chain drive, a belt drive, a V-belt drive, a flexible shaft, or by a combination of said gearings.

- 5 The action lever is preferably a part of a hand-brake lever. In another preferred embodiment of the invention, the action lever is part of a pedal, preferably of a pedal for a motor vehicle.

10 The pedal can preferably be a pedal, which can be adjusted in its dimensions to the user and wherein the action lever can be adjusted, so that the actuation force and the actuation path of the pedal remain constant despite the geometrical adjustment to the user. In that case the actuation conditions, i.e. the actuation feeling of the pedal remain constant independent from the geometrical adjustment.

- 15 In another embodiment the pedal can again preferably be a pedal, which can be adjusted in its dimensions to the user and wherein the action lever can be adjusted, so that the actuation force can also be adjusted to the user. In this case the actuation conditions, i.e. the actuation feeling of the pedal can be varied. The pedal can be adjusted to have a more soft or a more hard actuation feeling.

20 Preferably, the rotational shaft is independent from a geometrical adjustment means of the pedal. That means, that the geometrical adjustment is independent from the adjustment of the lever arm. In other words, for example the distance of a pedal to the user can be adjusted independently from its actuation feeling.

25 Preferably, a common actuation means is used for adjustment of the pedal to the user and for actuation of the first and second adjustment means.

30 In another embodiment at least two actuation means are used for adjustment of the pedal to the user and for actuation of the first and second adjustment means, wherein the actuation means are controlled by a control electronics.

- 7 -

Preferably, at least two of the pedals are arranged to form a pedal unit, wherein the first and second adjustment means of the action levers can be jointly driven for joint adjustment.

5

Only a single, common actuation means is preferably used for actuation of the first and second adjustment means of the pedals of a pedal unit.

Further preferred embodiments of the invention result from the subclaims.

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4. Short description of the drawings

In the following, the preferred embodiments of the present invention are described with reference to the drawings. Therein shows:

15 Fig. 1 a side view of a preferred embodiment of an action lever according to the invention, wherein the action lever is adjusted, so that it has shorter lever arms;

Fig. 2 a side view of the preferred embodiment of an action lever of fig. 1,
20 wherein the action lever has been adjusted, so that it has larger lever arms;

Fig. 3 a three-dimensional explosional view of the preferred embodiment of an action lever according to fig. 1;

25

Fig. 4 a three-dimensional view of the adjustment mechanism of an adjustable action lever, wherein the action lever arm is not shown;

Fig. 5 a three-dimensional view of a pedal according to a preferred embodiment
30 of the invention;

Fig. 6 a side view of another embodiment of the invention in the form of a pedal, which is geometrically adjustable to the driver;

Fig. 7 a side view of the pedal of fig. 6 during actuation.

5

5. Detailed description of the preferred embodiments

The present preferred embodiment describes an action lever according to the invention particularly for manual actuation, allowing a variation of the lever ratio. On the one hand, this is to make it possible to adjust the ratio of the action lever to
10 an individual user and on the other hand, if a geometrical adjustment of the action lever to a user has been effected, the ratio of the action lever can kept constant.

In figs. 1 to 4, a first embodiment of an action lever of the present invention is shown. The action lever consists substantially of an action lever arm 20, pivotably
15 supported in the fixed support 2 by means of a rotational shaft 15. On the action lever arm 20, a first force-inducing point 14 is provided, where a input force F_1 is passed into the action lever arm 20, as well as a second force-inducing point 16, where an output or reaction force F_2 is passed into the action lever arm 20. It is possible to induce tearing forces and forces of pressure as well as moments of
20 torsion at the shown force-inducing points. In the figs. 1 and 2 the forces F_1 and F_2 are represented by arrows at the reference numbers 14 and 16.

The force-inducing points 14 and 16 represent an arbitrary number of points on arbitrary positions of the arm, where various forces can be induced in the action
25 lever arm 20. For facilitating this illustration, it is supposed herein that the forces F_1 and F_2 are forces of pressure in the direction shown in figs. 1 and 2. The input force F_1 (at reference number 14) might e.g. be the force of pressure transmitted to a brake pedal by the foot of a driver. In the example of a brake pedal, the output force F_2 (at reference number 16) would be the force which is transmitted to the
30 main brake cylinder by the brake pedal.

- 9 -

In the example shown in fig. 1, the output force F_2 has a smaller lever action lever arm with regard to the rotational shaft 15 in comparison to the input force F_1 . If supposing for purpose of facilitation, that the rotational shaft 15 does not take up moments of torsion, and that the loss by friction etc. is small, it results that the
5 amount of the output force F_2 will exceed the amount of the input force F_1 . The amount of the input force F_1 multiplied with its resulting lever action lever arm equals to the output force F_2 multiplied with its resulting lever arm.

The resulting lever arms of the forces F_1 and F_2 are defined by the shortest distance between the symmetry axis of the rotational shaft 15 and the imaginary extension of the force vectors.
10

As mentioned above, the ratio of the forces F_1 and F_2 is its quotient or its relation. In the present embodiment, the ratio is the quotient of the distances of the lines of forces (the resulting lever arms) of F_1 and F_2 to the rotational shaft 15, which defines the rotation axis of the action lever arm 20. By relocating the rotational shaft 15 the distances to the lines of forces of F_1 and F_2 are varied. Thereby, also the quotient of this distances varies. That leads to a variation of the lever ratio of the action lever.
15

20 Fig. 2 shows in comparison to fig. 1 an action lever wherein the rotational shaft 15 was displaced upwardly. Thus, the lever arm of the output force F_2 and also the lever arm of the input force F_1 in relation to this rotational shaft are increased by the same amount. Since the amount of the output force F_2 is proportional to the relation of the lever arms of the forces F_1 and F_2 , this results in the output force F_2
25 in fig. 2 being smaller than the output force F_2 in fig. 1. Given that the same input force F_1 is induced into the action lever arm 20. A displacement of the rotational shaft 15 thus causes a variation of the translation ratio between the forces F_1 and F_2 .

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- 10 -

As it can be seen from fig. 1 and 2, the absolute position of the action lever arm 20 remains the same with regard to the fixed support 2. This is achieved by the rotational axis 15 being displaced within an elongated guide 18 of the fixed support and within an elongated guide 17 of the arm. In this embodiment the elongated guides are provided as elongated holes. Accordingly, the action lever arm 20 has to be supported in its longitudinal direction. This is effected by means of a first adjustment means 19, 21, preferably comprising an arm shaft 19 and first cam plates 21. The arm shaft 19 is rotably mounted within the action lever arm 20 and its orientation corresponds to the rotational shaft 15.

10

The arm shaft 19 is connected the first cam plates 21 and the arm shaft 19 rotates together with the first cam plates 21. The first cam plates 21 are provided with first cam slots 21a through which the rotational shaft 15 extends. Pivoting the first cam plates 21 results in displacing the rotational shaft 15 with regard to the action lever arm 20.

15

With regard to the fixed support 2, the rotational shaft 15 is displaced by means of a second adjustment means 22, 23. The second adjustment means 22, 23 comprises second cam plates 22 and at least one support pin 23. The rotational shaft 15 is connected to the second cam plates 22 so that the rotational shaft 15 rotates together with the second cam plates 22. The second cam plates 22 moreover comprise second cam slots 22a, into which the support pins 23 protrude, which are fixedly connected to the support 2. When pivoting the second cam plates 22, the position of the rotational shaft 15 can be continuously adjusted with relation to the fixed support 2.

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In this configuration, it is possible to pivot the first and the second cam plates 21, 22 simultaneously so that on the one hand, the rotational shaft 15 can be displaced in the elongated guides 17 and 18, which are here provided as elongated holes, and that on the other hand the position of the action lever arm 20 is not changed with relation to the support 2. When reducing the lever arms, the distanced from

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- 11 -

the rotational shaft 15 to the arm shaft 19 is increased, when increasing the lever arms, the distance from the rotational shaft 15 to the arm shaft 19 is reduced.

5 Since the first and the second cam plates 21, 22 are for the time being turned independently from each other, it is also possible to turn the first and the second cam plates 21, 22 not with the same amount, so that the position of the action lever arm 20 changes in relation to the mounting 2. Thereby, the resulting lever arms of the induced forces change, too.

10 The first and the second cam slots 21a, 22a preferably have the same shape, so that the same rotation of the first and the second cam plates 21, 22 results in the same displacement of the rotational shaft 15. As seen from the figs. 1 and 2, the first and the second cam plates 21, 22 comprise the same shape but are mounted twisted within the action lever.

15

Preferably, a rotation of the cam plates 21, 22 of 90° is sufficient to relocate the rotational shaft 15 by the necessary amount. However, also a smaller or larger rotation angle is possible if desired.

20 For displacing the rotational shaft 15, other devices than the afore-mentioned cam plates can be used. The rotational shaft 15 can e.g. be displaced via appropriate lever apparatuses or spindle drives.

25 As material for all elements of the action lever, various plastic materials or metals can be used. Preferably, the rotational shaft 15, the arm shaft 19, and the pin 23 are made of a steel. The fixed support 2 may also be preferably made of a sheet metal plate. The action lever arm 20 and the cam plates 21 and 22 are particularly preferably made of a plastic material, in particular of a glass-fiber reinforced plastic material.

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- 12 -

As shown in figs. 3 and 4, the cam plates 21 and 22 are each provided in duplicate. The cam plates 21 are provided within and the cam plates 22 outside a fixed support 2, which has in this embodiment the shape of a box.

5 For adjustment of the action lever, the first cam plates 21 and the second cam plates 22 can be driven. Preferably, this is effected by means of an electric motor. However, a manual drive is also possible. The cam plates 21, 22 can therein be driven via an arbitrary kind of gearing. Preferably, the cam plates 21, 22 are driven via a toothed wheel gearing, a spindle gearing, a cam gearing, a chain
10 drive, a belt drive, or a V-belt drive, a flexible shaft, or by a combination of said gearings. Therein, it is possible that motor and gearing are formed in that both cam plates 21, 22 are moved together. However, it is also possible to provide one gearing and one motor for each pair of the cam plates 21, 22. Then, independent adjustment is possible, thus making it is possible to change the length of the lever
15 20 if desired.

Fig. 5 shows another preferred embodiment of this invention, wherein the action lever is part of a pedal, preferably of a pedal for a motor vehicle. In this embodiment and in the embodiment for a hand brake lever, the brake way and the brake
20 force, to be applied by a driver, can be adjusted individually by means of the adjustable action lever. In the embodiment of fig. 5 the action lever arm 20 corresponds to the arm of the pedal and the force inducing point 14 of the input force F_1 corresponds to a foot-piece of the pedal.

25 Another embodiment results if a pedal adjustable to the user's seating position is used instead of the conventional pedal of fig. 5, as shown in figs. 6 and 7.

The pedal of figs. 6 and 7 can in addition to the adjustment of the lever arm also be geometrically adjusted. In this embodiment, the pedal's action lever arm consists of substantially only three parts, namely a subpedal 1, a positioning element
30 8 and a pedal foot 4. The subpedal 1 is pivotably mounted at the rotational shaft

- 13 -

15 of the ratio regulating mechanism. The pedal foot 4 is slideably arranged at the subpedal 1 by means of an elongated guide. The positioning element 8 is pivotably mounted at the subpedal 1 and slideably mounted at the pedal foot 4. By actuating the positioning element 8 or the pedal foot 4, the pedal foot 4 glides relatively to the subpedal 1 during a pivoting of the positioning element 8. The element, that has to be actuated, for example a braking cylinder, is connected to the subpedal 1.

Such an geometrical adjustable pedal is explained in detail in another patent application of the same applicant.

It is to be noted, that in this embodiment the ratio regulating mechanism is independent from the mechanism for the geometrical adjustment. In particular, the rotational shaft 15 is independently from the means for the geometrical adjustment of the pedal.

During the geometric adjustment of such a pedal, the action lever arm is usually changed. Thus, the actuating force and the actuating way are also changed, e.g. for actuating the brake. Thus, a small driver, whose brake pedal is adjusted to be long, has to apply a larger actuating way to the foot piece of the pedal in order to create the same braking force in comparison to the short actuation way of a short pedal. This effect can be compensated by an adjustable action lever according the invention. Thus, actuation force and actuation way (the actuation feeling) of the pedal are constant despite the geometric adjustment to the user.

On the other hand it may particularly desired to adjust the actuation force and actuation way (the actuation feeling) of the pedal. In that case the ratio adjustment mechanism can be used to perform such a function. The pedal can then be adjusted to have a more soft or a more hard actuation feeling.

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- 14 -

The geometric adjustment of the pedal and the adjustment of the lever action lever arm are preferably done simultaneously. A common actuation means (e.g. an electric motor) can be used when an appropriate gearing is used. However, it is of course also possible to use a separate actuation means for each possibility of adjustment. The lever action lever arm can then be adjusted independently of the adjustment to the driver. However, if now a compensation of actuation force and actuation way has to be effected, the actuation means have to be controlled by a control electronics, so that for each desired position of the pedal the appropriate position of the compensation is selected.

10

Fig. 7 shows, how the sub pedal 1, which here corresponds to the action lever arm 20, rotates around the rotational shaft 15 during the actuation of the pedal. It can also be seen, that the first adjustment means 19, 21 rotates together with the sub-pedal 1 or the action lever arm 20 during actuation.

15

In another embodiment of the invention, at least two pedals are grouped into a pedal unit, wherein the adjustment means of the different action levers of the pedals are jointly driven for even adjustment. Preferably, a single common adjustment element is used, e.g. a motor with a gearing.

20

List of reference numbers

- | | |
|----|---|
| 1 | subpedal |
| 2 | fixed support |
| 4 | pedal foot |
| 25 | 8 positioning element |
| 14 | first force-inducing point |
| 15 | rotational shaft |
| 16 | second force-inducing point |
| 17 | elongated guide of the arm |
| 30 | 18 elongated guide of the fixed support |
| 19 | arm shaft |

- 15 -

- 20 action lever arm
- 21 first cam plates
- 21a first cam slots
- 22 second cam plates
- 5 22a second cam slots
- 23 support pin(s)

Patent Claims

1. Ratio regulating mechanism for a manually actuated action lever, in particular for the use in a motor vehicle, comprising:
 - 5 a. a mounting (2);
 - b. an action lever arm (20);
 - 10 c. a rotational shaft (15) for rotably mounting of the action lever arm (20) at the mounting (2), wherein the rotational shaft (15) is relocatably mounted at the action lever arm (20) and relocatably mounted at the mounting (2);
 - d. a first adjustment means (19, 21) for the relocation of the rotational shaft
15 (15) in relation to the action lever arm (20); and
 - e. a second adjustment means (22, 23) for the relocation of the rotational shaft (15) in relation to the mounting (2).
- 20 2. Ratio regulating mechanism in accordance with claim 1, wherein the first adjustment means (19, 21) and the second adjustment means (22, 23) are adjustable, so that during the relocation of the rotational shaft (15) the position of the action lever arm (20) in relation to the mounting (2) is maintained.
- 25 3. Ratio regulating mechanism in accordance with one of the claims 1 or 2, wherein the relocation of the rotational shaft (15) by said first adjustment means (19, 21) occurs in an opposite direction to the relocation of the rotational shaft (15) by said second adjustment means (22, 23).

- 17 -

4. Ratio regulating mechanism in accordance with one of the claims 1 - 3,
wherein said first adjustment means (19, 21) and said second adjustment
means (22, 23) are simultaneously actuated.
- 5 5. Ratio regulating mechanism in accordance with one of the claims 1 - 4,
wherein the first adjustment means (19, 21) is mounted at the action lever arm
(20).
6. Ratio regulating mechanism in accordance with one of the claims 1 - 5,
10 wherein the second adjustment means (22, 23) is mounted at the mounting (2).
7. Ratio regulating mechanism in accordance with one of the claims 1 - 6,
wherein the action lever arm (20) comprises an elongated guide (17) and the
mounting (2) comprises an elongated guide (18) for the relocatable mounting
15 of the rotational shaft (15).
8. Ratio regulating mechanism in accordance with one of the claims 1 - 7,
wherein the first adjustment means (19, 21) comprises an arm shaft (19),
which is supported in the action lever arm (20), and first cam plates (21),
20 which are connected to the arm shaft (19) and the rotational shaft (15), so that
a relocation of the rotational shaft (15) in relation to the action lever arm (20)
results from a rotation of the first cam plates (21).
9. Ratio regulating mechanism in accordance with one of the claims 1 - 8,
25 wherein the second adjustment means (22, 23) comprises a support pin (23),
which is connected to the mounting (2), and second cam plates (22), which are
connected to the support pin (23) and the rotational shaft (15), so that a relo-
cation of the rotational shaft (15) in relation to the mounting (2) results from a
rotation of the second cam plates (22).

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- 18 -

10. Ratio regulating mechanism in accordance with one of the claims 8 or 9,
wherein the first cam plates (21) comprise first cam slots (21a), through which
the rotational shaft (15) extends, and the second cam plates (22) comprise sec-
ond cam slots (22a), through which the support pin (23) extends.
- 5
11. Ratio regulating mechanism in accordance with claim 10, wherein said rota-
tional shaft (15) is attached to said second cam plates (22) and slideably ar-
ranged through said first cam slots (21a), so that the rotational shaft (15) is
functional connected to both adjustment means (19, 21, 22, 23).
- 10
12. Ratio regulating mechanism in accordance with one of the claims 10 or 11,
wherein the first and the second cam slots (21a, 22a) have substantially the
same shape and length.
- 15
13. Ratio regulating mechanism in accordance with one of the claims 8 - 12
wherein the first and the second cam plates (21, 22) are rotated by the same
rotation angle during adjustment of the rotational shaft (15).
14. Ratio regulating mechanism in accordance with one of the claims 8 - 13,
20 wherein the first cam plates (21) and/or the second cam plates (22) are driven
by means of an electric motor.
15. Ratio regulating mechanism in accordance with one of the claims 8 - 13,
wherein the first cam plates (21) and/or the second cam plates (22) are manu-
25 ally driven.
16. Ratio regulating mechanism in accordance with one of the claims 14 or 15,
wherein the first cam plates (21) and/or the second cam plates (22) are driven
either by means of a toothed wheel gearing, a spindle gearing, a cam gearing,
30 a chain drive, a belt drive, or a V-belt drive, a flexible shaft or by a combina-
tion of said gearings.

- 19 -

17. Ratio regulating mechanism in accordance with one of the claims 1 - 16, wherein the ratio regulating mechanism is part of a hand-brake lever.
- 5 18. Ratio regulating mechanism in accordance with one of the claims 1 - 16, wherein the ratio regulating mechanism is part of a pedal, preferably of a pedal for a motor vehicle.
- 10 19. Ratio regulating mechanism in accordance with claim 18, wherein the pedal is a pedal which can be adjusted in its dimensions to the user and wherein the action lever can be adjusted, so that the actuation force and the actuation path of the pedal remain constant despite the geometrical adjustment to the user.
- 15 20. Ratio regulating mechanism in accordance with claim 18, wherein the pedal is a pedal which can be adjusted in its dimensions to the user and wherein the action lever can be adjusted, so that the actuation force can be adjusted to the user.
- 20 21. Ratio regulating mechanism in accordance with one of the claims 19 or 20, wherein the rotational shaft (15) is independent from a geometrical adjustment means of the pedal.
- 25 22. Ratio regulating mechanism in accordance with one of the claims 18 - 21, wherein a common actuation means is used for geometrical adjustment of the pedal to the user and for actuation of the first (19, 21) and second adjustment means (22, 23).
- 30 23. Ratio regulating mechanism in accordance with claim 18 - 21, wherein at least two actuation means are used for geometrical adjustment of the pedal to the user and for actuation of the first (19, 21) and second adjustment means (22, 23), wherein the actuation means are controlled by a control electronics.

- 20 -

24. Ratio regulating mechanism in accordance with one of the claims 18 - 23,
wherein at least two of the pedals are arranged to form a pedal unit, wherein
the first (19, 21) and second adjustment means (22, 23) of the action levers
5 can be jointly driven for joint adjustment.

25. Ratio regulating mechanism in accordance with claim 24, wherein only a sin-
gle, common actuation means is used for actuation of the first (19, 21) and
second adjustment means (22, 23).

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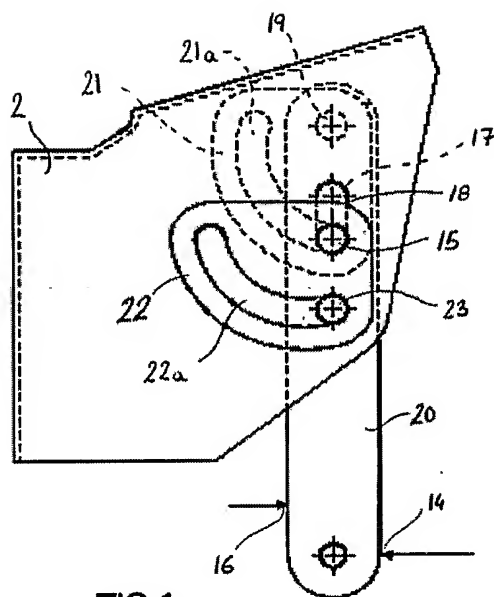


FIG. 1

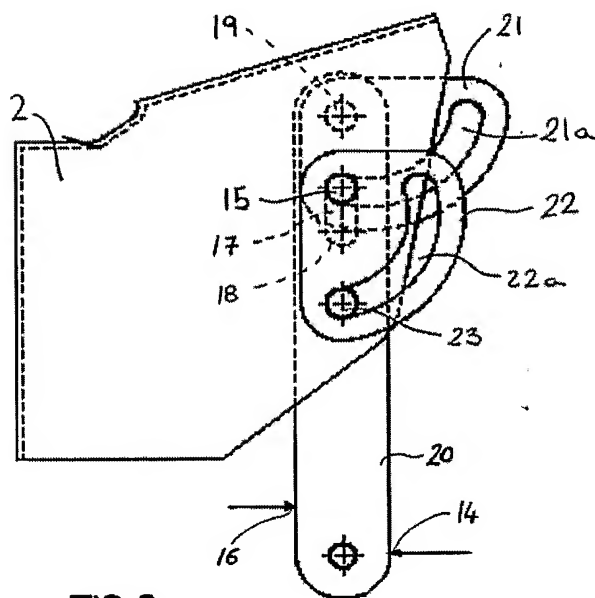


FIG. 2

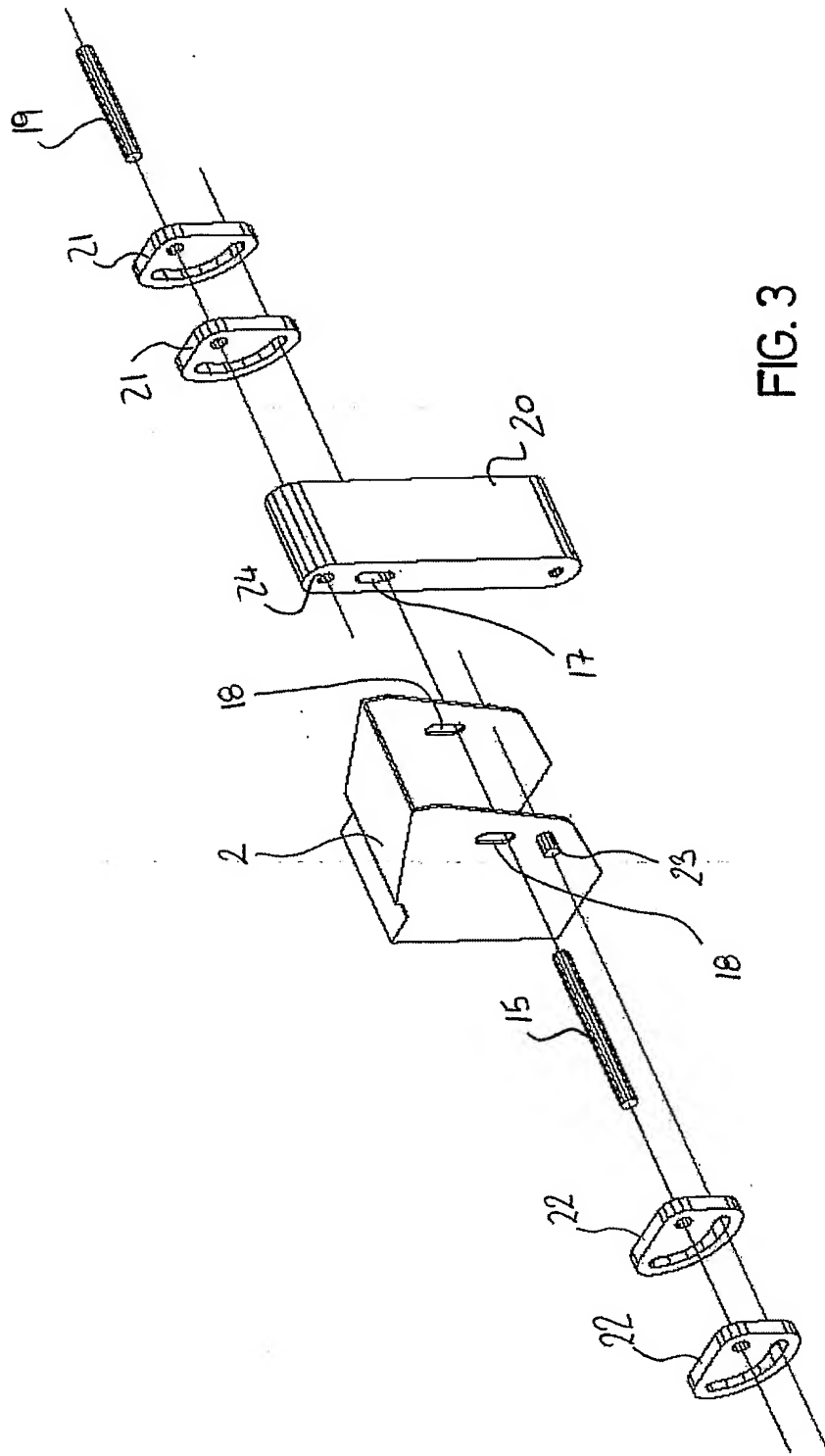
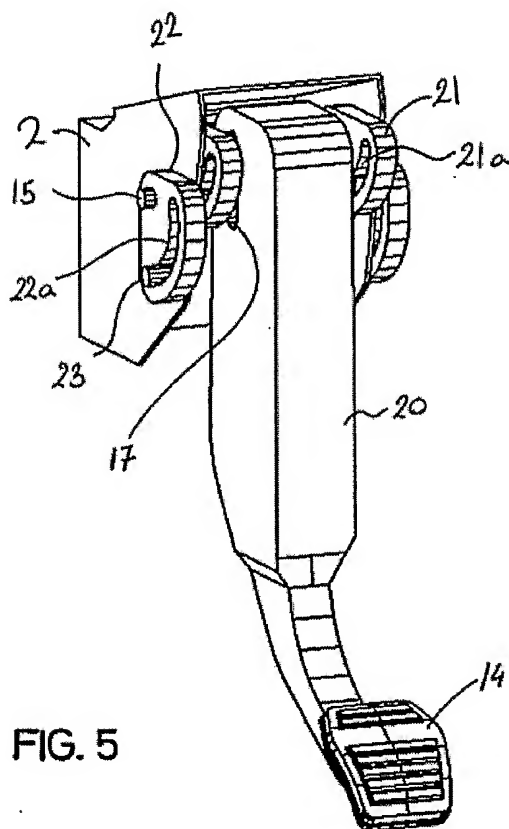
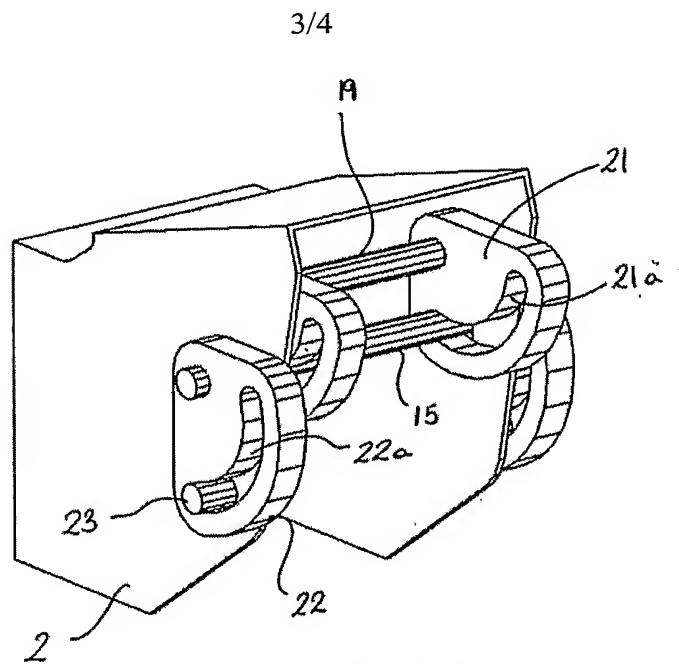


FIG. 3



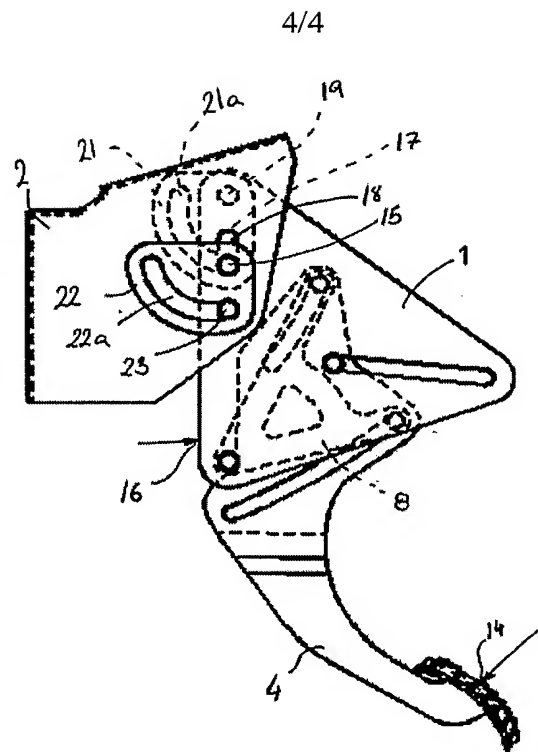


FIG. 6

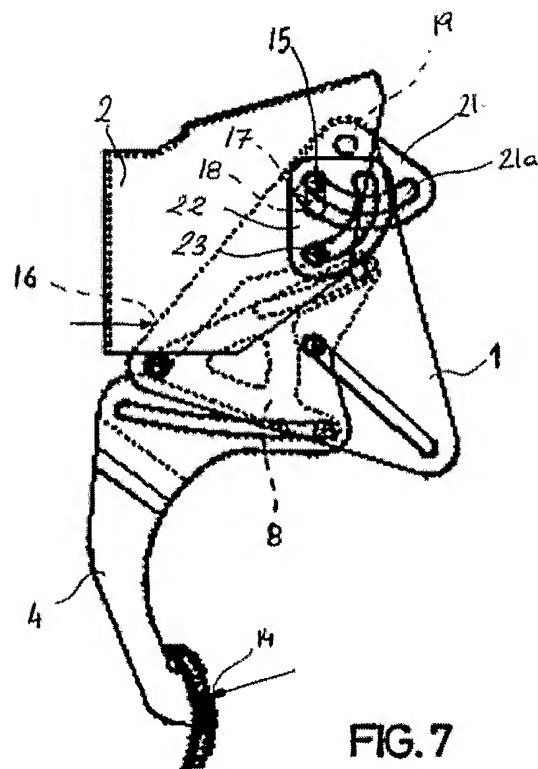


FIG. 7

INTERNATIONAL SEARCH REPORT

International Application No

PCT/EP 03/13177

A. CLASSIFICATION OF SUBJECT MATTER IPC 7 B60T7/10 B60T7/06 B60T11/08 G05G7/04		
According to International Patent Classification (IPC) or to both national classification and IPC		
B. FIELDS SEARCHED Minimum documentation searched (classification system followed by classification symbols) IPC 7 B60T G05G B60K		
Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched		
Electronic data base consulted during the international search (name of data base and, where practical, search terms used) EPO-Internal		
C. DOCUMENTS CONSIDERED TO BE RELEVANT		
Category, °	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X	US 4 989 474 A (CICOTTE EDMOND B ET AL) 5 February 1991 (1991-02-05) column 4, line 32 -column 5, line 3; figure 2 column 7, line 44 -column 8, line 15 ---	1,7,8, 14,18-20
X	US 5 460 061 A (REDDING HARRY L ET AL) 24 October 1995 (1995-10-24) column 3, line 42 -column 4, line 15 column 5, line 31 - line 44 ---	1,5-7, 14, 18-20, 24,25
A	FR-2 739 947-A-(RENAULT) --- 18 April 1997 (1997-04-18) abstract; figures 1,2 --- <div style="text-align: center;">-/-</div>	1,24,25
<div style="display: flex; justify-content: space-between;"> <input checked="" type="checkbox"/> Further documents are listed in the continuation of box C. <input checked="" type="checkbox"/> Patent family members are listed in annex. </div>		
° Special categories of cited documents :		
<div style="display: flex; justify-content: space-between;"> <div style="width: 45%;"> <p>*A* document defining the general state of the art which is not considered to be of particular relevance</p> <p>*E* earlier document but published on or after the international filing date</p> <p>*L* document which may throw doubts on priority claim(s) or which is cited to establish the publication date of another citation or other special reason (as specified)</p> <p>*O* document referring to an oral disclosure, use, exhibition or other means</p> <p>*P* document published prior to the international filing date but later than the priority date claimed</p> </div> <div style="width: 45%;"> <p>*T* later document published after the international filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention</p> <p>*X* document of particular relevance; the claimed invention cannot be considered novel or cannot be considered to involve an inventive step when the document is taken alone</p> <p>*Y* document of particular relevance; the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such documents, such combination being obvious to a person skilled in the art.</p> <p>*&* document member of the same patent family</p> </div> </div>		
Date of the actual completion of the international search <div style="text-align: center; font-weight: bold;">31 March 2004</div>		Date of mailing of the international search report <div style="text-align: center; font-weight: bold;">06/04/2004</div>
Name and mailing address of the ISA European Patent Office, P.B. 5818 Patentlaan 2 NL - 2280 HV Rijswijk Tel. (+31-70) 340-2040, Tx. 31 651 epo nl, Fax: (+31-70) 340-3016		Authorized officer <div style="text-align: center; font-weight: bold;">Ibáñez Lajo, M</div>

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C.(Continuation) DOCUMENTS CONSIDERED TO BE RELEVANT

Category *	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
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A	EP 1 251 417 A (DELPHI TECH INC) 23 October 2002 (2002-10-23) abstract; figure 1 ---	1
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